

Effects of the extrusion process parameter

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ABSTRACT

In this study finite element analyses of rod extrusion process are carried out considering various processing parameters. The objective is to study the effect of these parameter on mechanical properties. Eighty one cases are simulated and corresponding stress, plastic strain and load distribution are critically examined. Based on these results, salient conclusions are drawn.

Key words: Extrusion, finite element analysis, die angle, stress, hardening exponent.

INTRODUCTION

Extrusion is an important Metal forming operation. It is a manufacturing process used to create long objects of a fixed cross sectional profile. The extrusion process is based on the plastic deformation of a material due to compressive and shears forces only. No tensile forces are applied to the extruded metal. The latter allows the material to withstand high deformation without tearing out the material. Basically, this procedure is based on the reducing and shaping the cross section of piece of metal squeezing the material through an orifice or a die. Typically the blocks of metal used for this procedure are long straight parts with circular cross sections.

The extrusion of metals is used for the minimized need of extra finishing tasks the parts need after production; extruded Parts usually have a constant cross-section along its span.

An analysis done by Kang¹, of three dimensional hot extrusion processes through landless square die is carried out as a non steady state problem. In the problem, difficulties arise from

the severe distribution and die interference of elements at the aperture rim of the die, even with a small punch travel. Finite element simulation is impossible without intermittent remeshing procedures. To overcome these difficulties, an automatic remeshing technique is proposed by employing a modular conceptual mesh structure. Venkata et. al [2] carried out a comprehensive investigation of an axisymmetric steady state tube extrusion through a streamlined die is carried out by the FEM to study the influence of process variable on tool design and final product quality for a strain hardening material. The process variable considered is: The reduction in area, coefficient of friction, mandrel radius, dies length, the hardening capacity of material. The extrusion parameter studied is: the extrusion pressure, die pressure, mandrel pressure, hydrostatic stress distribution, strain rate distribution, strain distribution. Hur *et.al*³ carried out an analysis for the dimensional accuracy of the cold forged products that is strongly dependent on the elastic characteristics of the die.

Gouveia *et.al.*,⁴ carried out the finite element modeling of cold formed extrusion. Simulating metal forming processes using an

updated Lagrangian finite element formulation is not ideal when steady state material flow conditions prevail. Firstly, repeated calculation of large non linear finite element system are needed for continuously updating the mesh and secondary remeshing operations must be undertaken to avoid excessive mesh distortion and to introduce localized refinements in regions where large gradients are likely to occur.

Cosenza *et al.*,⁵ carried out the damage and fracture study of cold extrusion dies. In this, die fracture in cold extrusion was investigated considering a few different die reduction zone geometries.

Peng *et al.*,⁶ carried out the finite element analysis of spring back and secondary yielding effect during forward extrusion. The response of work material during forward extrusion and the subsequent unloading process was analyzed with a view to examining difficulties in prediction of component form errors, when different consecutive models are used.

Geometrical, material & frictional parameters

Following geometrical, material and frictional parameters are considered in the study -

Geometrical properties

Billet diameter = 80 mm
Billet Length = 60 mm

Extruded Rod Diameter = 40 mm

Half Die angle = 30°, 45° and 60°

In this way, extrusion ratio comes out to be 4.

Material properties

Billet is modeled as rigid material. Power law equation has been used for the modeling the stress strain behavior:

$$\sigma = K \varepsilon^n$$

Where n is strain hardening exponent and K is strength coefficient.

Following values of material parameters are accounted-

Young's Modulus = 78000 MPa

Poisson's ratio = 0.33

Strength coefficient = 500, 600 and 700 MPa

Strain hardening exponent n = 0.1, 0.15 and 0.2

Friction

Coulomb friction criteria have been taken into the account for contact modeling. Three values of Coulomb friction coefficient viz 0.1, 0.15 and 0.2 are taken into the account.

FE modeling and boundary conditions

Finite Element Modeling of the rod extrusion process is carried out using MSC. Marc preprocessor. Die and punch are modeled as rigid and billet is modeled as deformable bodies. Axis symmetric finite element modeling is carried out using 4 nodes quadrilateral elements. Typical FE model is shown in Fig.2. There are 600 element

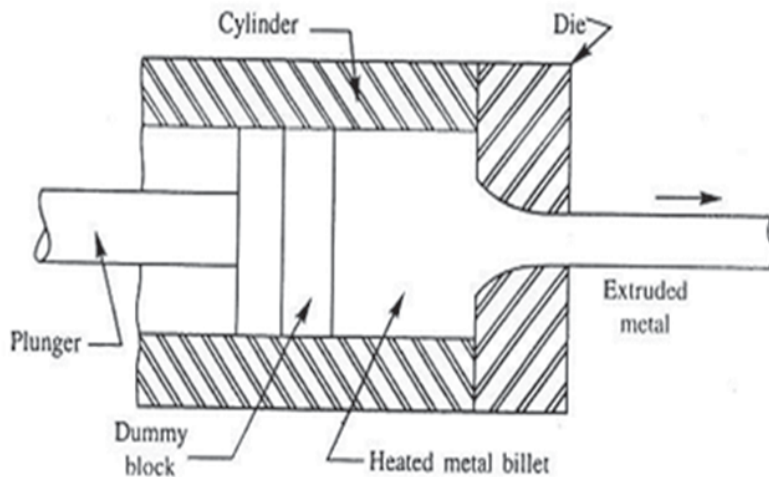


Fig. 1: Schematic of Extrusion

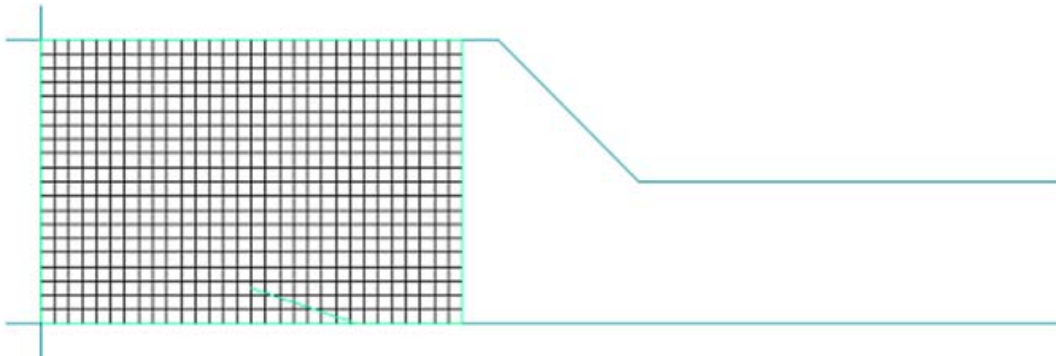


Fig. 2: FE Mesh Generation

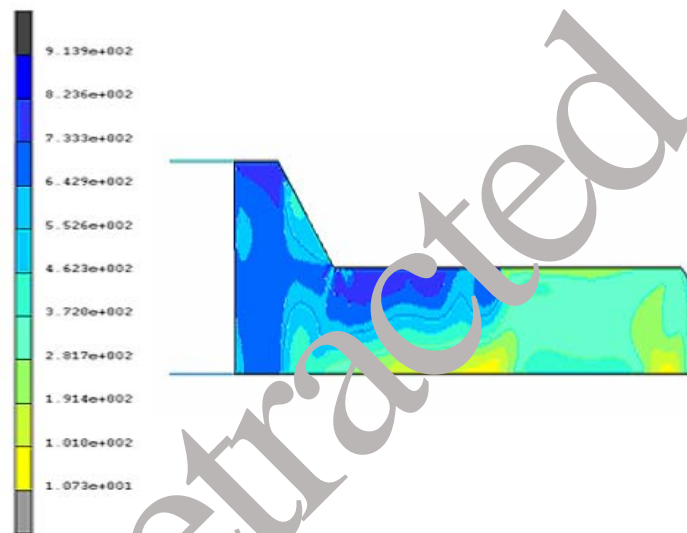


Fig. 3: Effective stress contour (MPa)

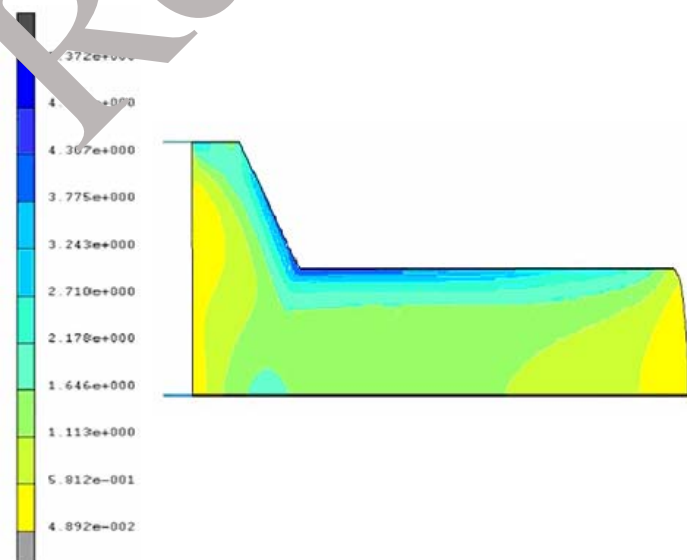


Fig. 4: Plastic Strain Contour

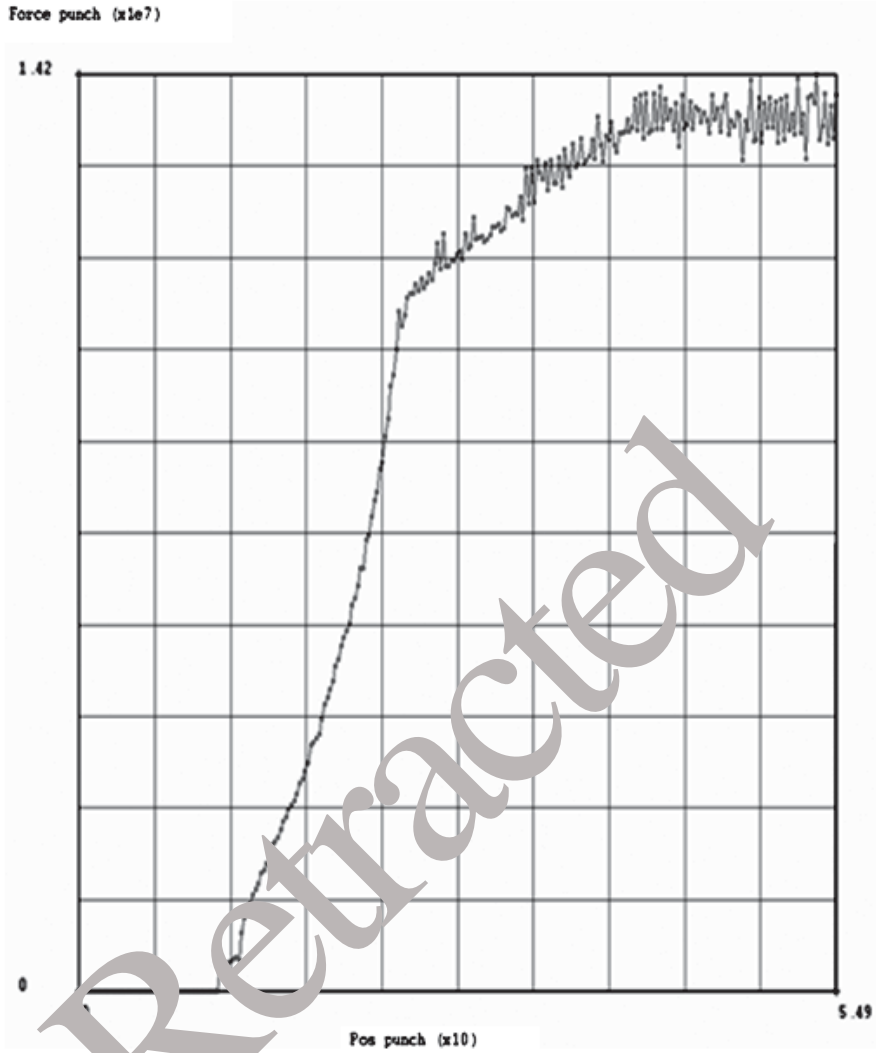


Fig. 5: Load Distribution curve

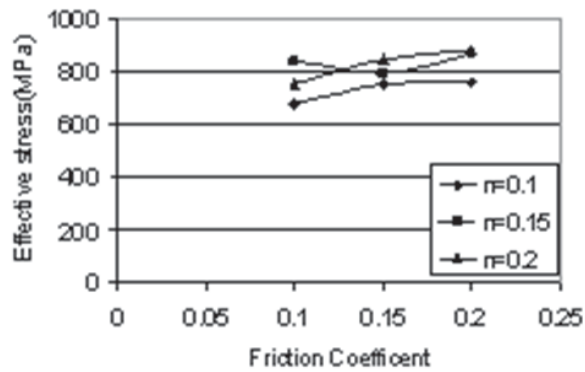


Fig. 6: Effective stress vrs friction ($K = 600, \theta = 30^\circ$)

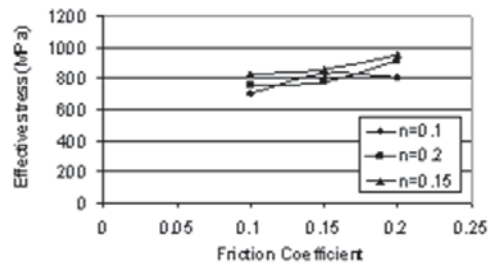


Fig. 7: Effective stress vrs friction ($K = 600, \theta = 45^\circ$)

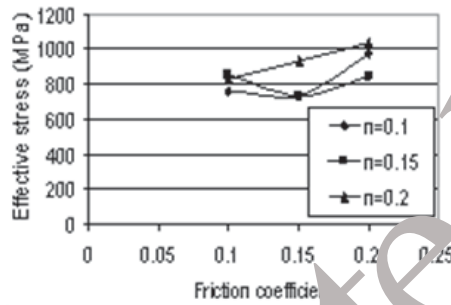


Fig. 8: Effective stress vrs friction ($K = 600, \theta = 60^\circ$)

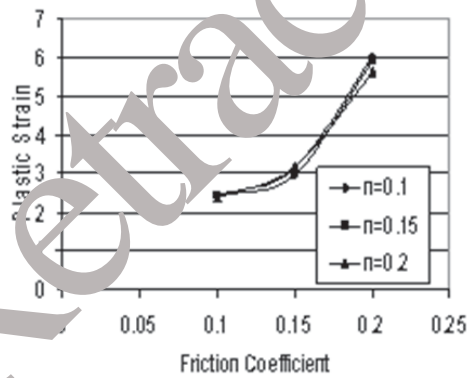


Fig. 9: Plastic strain vrs friction ($K = 600, \theta = 30^\circ$)

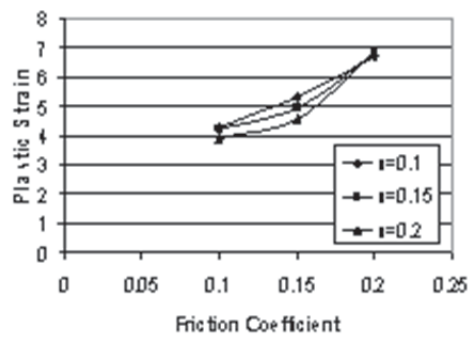


Fig. 10: Plastic strain vrs friction ($K = 600, \theta = 45^\circ$)

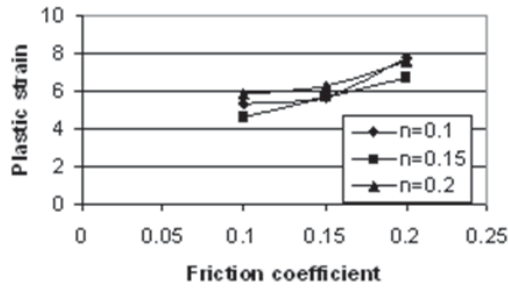


Fig. 11: Plastic strain vs friction (K = 600, θ = 60°)

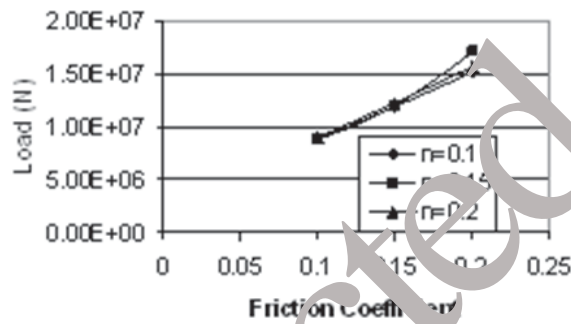


Fig. 12: Load vs friction (K = 600, θ = 30°)

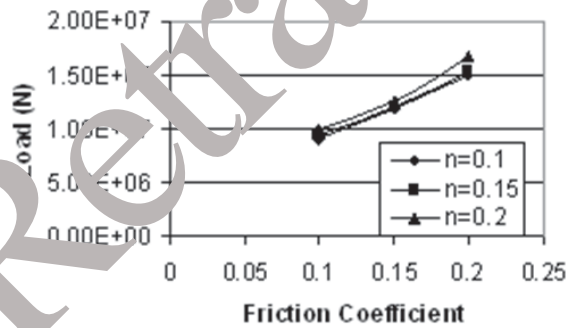


Fig. 13: Load vs friction (K = 600, θ = 45°)

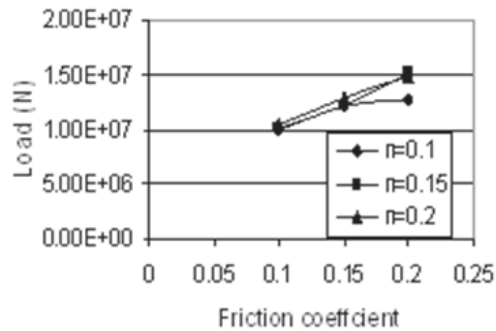


Fig. 14: Load vs friction (K = 600, θ = 60°)

and 651 nodes in the model. Displacement boundary condition is applied on the ram. Interaction of billet and die is accounted through contact algorithms of the software. Finite element simulation is carried out in incremental manner in 50 steps. Ram is given a displacement of 20 mm. Simulation results for the three dies, considering different strength factor, can be critically examined as below.

RESULTS AND DISCUSSION

Effective stress distribution

A typical stress contour in the extruded rod is shown in Fig. 3. It can be observed that quite large stress appears in the extrusion process. First

of all keeping $K=500$ MPa, we vary the Die angle as 30 , 45 and 60 as shown in Table 1, Table 2 and Table 3. We observe that as friction coefficient increases the value of effective stress increases with increasing value of Hardening Exponent. As the Die angle increases the slope of curve decreases.

Again for $K = 600$ and 700, we plot the graph considering the angle 30 , 45 and 60 one by one. And a close study is done by plotting graph how the Effective stress varies when other parameters are varies.

Plastic strain distribution

A typical plastic strain contour is shown in

Table 1: Stress, strain & load for 30 Die

S. No.	K	n	Friction coulomb	Effective stress (N/mm ²)	Plastic strain	Load (N)
1	500	0.1	0.1	558.6	2.438	7.27E+06
2	500	0.15	0.1	627.1	2.389	7.30E+06
3	500	0.2	0.1	657.1	2.452	7.37E+06
4	500	0.1	0.15	615	3.088	9.93E+06
5	500	0.15	0.15	648.1	3.098	9.85E+06
6	500	0.2	0.15	677.8	3.182	1.01E+07
7	500	0.1	0.2	644	6.172	1.22E+07
8	500	0.15	0.2	688.2	5.648	1.25E+07
9	500	0.2	0.2	733	5.567	1.28E+07
10	600	0.1	0.1	672.26	2.433	8.72E+06
11	600	0.15	0.1	835.2	2.434	8.86E+06
12	600	0.2	0.1	749.8	2.393	8.90E+06
13	600	0.1	0.15	749.9	3.119	1.19E+07
14	600	0.15	0.15	789.7	2.979	1.19E+07
15	600	0.2	0.15	841.5	3.168	1.23E+07
16	600	0.1	0.2	754.8	6.037	1.51E+07
17	600	0.15	0.2	863.8	5.886	1.70E+07
18	600	0.2	0.2	875.2	5.615	1.57E+07
19	700	0.1	0.1	812.6	2.447	1.03E+07
20	700	0.15	0.1	883.3	2.38	1.04E+07
21	700	0.2	0.1	886.5	2.43	1.06E+07
22	700	0.1	0.15	872.4	3.054	1.38E+07
23	700	0.15	0.15	946.2	3.072	1.43E+07
24	700	0.2	0.15	969	3.056	1.43E+07
25	700	0.1	0.2	890.9	5.9	1.78E+07
26	700	0.15	0.2	956.6	5.726	1.96E+07
27	700	0.2	0.2	1025	5.144	1.93E+07

Fig. 4. It can be observed that strain can be as high as 7.

Plastic strain has been studied under the different condition of frictional coefficient, Die angle, Strength Factor and hardening Exponent.

For $K = 500$ and $\theta = 30, 45$ and 60 the value of frictional coefficient increases plastic strain also increases or from increasing value of Hardening exponent slope of curve increases. For higher value of Die angle slope of plastic strain changes from minimum to maximum. For $K = 600$ and Die angle 30 for all value of n , strain increases with increasing value of n . For 45 and 60 slope reduces with n and increasing value of n .

As die angle increases plastic strain increases for constant value of strength factor as observed in table. While for constant die angle the strength factor increases slope increases, but for increasing value of die angle slope of Plastic Strain decreases as observed. With increasing value of hardening exponent plastic strain increases.

Load distribution

A typical load curve of the extruded rod is shown in Fig. 5. It can be observed that quite large load appears in the extrusion process.

The applied load on billet depends on various parameters like frictional coefficient, strength factor, Hardening exponent and die angle. In all these frictional force plays most important role.

Table 2: Stress, strain and load for 45 Die

S. No.	K	n	Friction coulomb	Effective stress (N/mm ²)	Plastic strain	Load (N)
1	500	0.1	0.1	688.7	4.292	7.55E+06
2	500	0.15	0.1	688.1	4.386	7.58E+06
3	500	0.2	0.1	735.6	4.032	8.00E+06
4	500	0.1	0.15	671.5	5.371	9.55E+06
5	500	0.15	0.15	726.1	4.997	9.74E+06
6	500	0.2	0.15	718.6	4.653	1.04E+07
7	500	0.1	0.2	689.1	6.747	1.17E+07
8	500	0.15	0.2	769.1	7.14	1.27E+07
9	500	0.2	0.2	835.2	6.586	1.23E+07
10	600	0.1	0.1	707.9	4.264	9.19E+06
11	600	0.15	0.1	761.2	4.221	9.48E+06
12	600	0.2	0.1	824.5	3.915	9.92E+06
13	600	0.1	0.15	837	5.329	1.19E+07
14	600	0.15	0.15	781.8	4.949	1.21E+07
15	600	0.2	0.15	861.6	4.553	1.26E+07
16	600	0.1	0.2	810.1	6.724	1.50E+07
17	600	0.15	0.2	912.8	6.853	1.53E+07
18	600	0.2	0.2	955.9	6.869	1.67E+07
19	700	0.1	0.1	874	4.368	1.09E+07
20	700	0.15	0.1	923.1	3.889	1.12E+07
21	700	0.2	0.1	1125	3.858	1.15E+07
22	700	0.1	0.15	913.9	5.372	1.42E+07
23	700	0.15	0.15	965.8	4.956	1.49E+07
24	700	0.2	0.15	1058	4.659	1.57E+07
25	700	0.1	0.2	930.9	7.12	1.92E+07
26	700	0.15	0.2	995	7.096	2.19E+07

Table 3: Stress, strain & load for 60 Die

S. No.	K	n	Friction coulomb	Effective stress (N/mm ²)	Plastic strain	Load (N)
1	500	0.1	0.1	735.8	5.953	7.88E+06
2	500	0.15	0.1	708.9	5.764	8.32E+06
3	500	0.2	0.1	775.7	4.507	8.81E+06
4	500	0.1	0.15	711.9	5.627	9.90E+06
5	500	0.15	0.15	718.1	5.376	9.67E+06
6	500	0.2	0.15	924.8	5.935	9.70E+06
7	500	0.1	0.2	672.2	8.06	9.51E+06
8	500	0.15	0.2	888.1	8.776	1.05E+07
10	600	0.1	0.1	759	5.27	9.90E+06
11	600	0.15	0.1	854.7	4.588	1.02E+07
12	600	0.2	0.1	828.1	5.835	1.04E+07
13	600	0.1	0.15	730.2	5.614	1.21E+07
14	600	0.15	0.15	842.7	5.743	1.20E+07
15	600	0.1	0.2	972.5	7.778	1.27E+07
16	600	0.15	0.2	844.1	6.784	1.51E+07
17	600	0.2	0.2	1032	7.602	1.48E+07
18	700	0.1	0.1	917.7	5.076	1.16E+07
19	700	0.15	0.1	1113	4.542	1.22E+07
20	700	0.2	0.1	1127	5.031	1.30E+07
21	700	0.1	0.15	1084	5.984	1.44E+07
22	700	0.2	0.15	1040	5.678	1.53E+07
23	700	0.1	0.2	911.6	7.274	1.85E+07
24	700	0.15	0.2	1224	7.639	1.91E+07
25	700	0.2	0.2	1145	7.723	2.03E+07

As friction increases the work load increases and thus applied Load. Since friction occurs only at the surface and throughout the thickness of metal billet. It introduces microscopic inhomogeneity, resulting in micro cracks on surface and weaker products having lower fatigue strength.

Variation of Load distribution with the variation of frictional Coefficient has been shown from Fig 13 to Fig. 14, changing with various parameters. For K = 500, we vary angle and observe that the load curve slope increases with the increases in value of hardening exponent. In case die angle 60 initially load increases but after a certain value of friction coefficient load decreases with increasing value of hardening exponent.

Again for K = 600 and 700, we plot the graph considering the angle 30, 45 and 60 one

by one. And a close study is done by plotting graph how the Effective stress varies when other parameters are varies.

In the same way when angle die are constant and strength factor increases then it goes towards constant and the load is most affected by friction coefficient. As friction coefficient increases, load increases, it is desirable to keep the load requirement as low as possible.

CONCLUSIONS

In this study effects of processing parameters on rod extrusion are studied using FEM. 81 cases considering various geometrical, material and friction parameters are simulated.

1. For constant die angle, stresses are proportional and inversely proportional to

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|---|---|
| <p>hardening exponent and strength coefficient respectively.</p> <p>2. Stress decreases with increase in die angle.</p> <p>3. Plastic strain decreases with increase in strain hardening exponent.</p> <p>4. Plastic strain decreases with the increasing value of Die angle.</p> | <p>5. Plastic strain increases with increasing value of strength Factor.</p> <p>6. Load increases with the increasing value of strain hardening Exponent.</p> <p>7. Load decreases with increasing value of strength factor.</p> <p>8. Load increases with increasing value of Die angle.</p> |
|---|---|

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